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Question Paper Code: 90859

B.E./B.Tech. DEGREE EXAMINATIONS, NOVEMBER/DECEMBER 2022.

Fourth/Sixth Semester

Mechanical Engineering

ME 8493 — THERMAL ENGINEERING — I

(Common to Mechanical Engineering (Sandwich))

(Regulations 2017)

Time: Three hours

Maximum: 100 marks

Answer ALL questions.

PART A — $(10 \times 2 = 20 \text{ marks})$

- 1. Draw the P-V and T-s diagram for Diesel cycle.
- 2. List the four processes involved in air standard Brayton cycle.
- 3. How intercooling increases efficiency of compressor?
- 4. Differentiate between reciprocating and centrifugal compressor.
- 5. List the geometric differences between the 4-stroke and 2-stroke SI engine.
- 6. What are the desirable properties of fuel for CI engine?
- 7. Why multi point fuel injection system is needed for SI engines?
- 8. What is the need for emission norms? Name the emission norm followed now.
- 9. How performance of stationary gas turbine can be improved?
- 10. Why special alloys are used for turbine blades in a gas turbine?

PART B — $(5 \times 13 = 65 \text{ marks})$

11.	(a)	12 t	n air-standard Otto cycle, the pressure at the end of compression imes that at the start, the temperature of air at the beginning	gof
			pression is 30°C and maximum temperature attained in the cycle 0°C. Determine	9 1S
		(i)	Compression ratio	(4)
		(ii)	Thermal efficiency	(4)
		(iii)	Work done.	(5)
			Or	
	(b)	dual temp pres	compression ratio and expansion ratio of an oil engine working cycle are 9 and 5 bar respectively. The initial pressure a perature of the air are 1 bar and 30°C. The heat liberated at constant sure is twice the heat liberated at constant volume. The expans compression follow the law $PV^{1.25}$ = constant. Determine	and ant
		(i)	Pressure and temperature at all salient points	(7)
		(ii)	Mean effective pressure.	(6)
12.	(a)	300 drive 1 ba deliv Assu	ingle cylinder, single acting reciprocating air compressor runs rpm, is driven by a 23 kW electric motor. Mechanical efficiency of e between motor and compressor is 87%. Air inlet conditions ar and 15 °C and the delivery pressure is 8 bar. Calculate the free very, volumetric efficiency, bore and stroke of the compressume that the index of compression and expansion as 1.3, rance volume is 7% of swept volume and that the bore is equal to ke.	the are air sor.
			alver not said backman Or, or beviewed assessing and end tack	
	(b)	low pres 4 ba cylin stro Assi	ingle acting two stage air reciprocating compressor delivers air par. The temperature and pressure of air before the compression pressure cylinder are 25°C and 1 bar. The discharge pressure of leaving the inter-coole are and air is cooled to 25°C. The diameter and stroke of low-pressinder are 40 cm and 50 cm respectively. The clearance volume is 5% ke volume in both the cylinders. The speed of compressor is 200 recuming the index of compression and re-expansion in both anders as 1.25 and Cp = 1.005 kJ/kg K, determine	ow- or is ure 6 of om.
		(i)	Power required to run the compressor	(7)
		(ii)	Heat rejected in the intercooler per minute.	(6)
13.	(a)		cribe about the construction of 4-stroke SI engine with a neat sket	ch.
	on be		multiple more auto-sala en la Orimon agias ma vol Room esta el tagiw	

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discuss the factors responsible for knocking in Cl engines.

(b)

Describe about the factors responsible for knocking in SI engines. Also

14.	(a)	With a neat sketch, brief about the construction of common rail direct injection system.
		Or
	(b)	For a multi-cylinder engine, elaborate on the construction of water based cooling system employed in engines with supporting sketches.
15.	(a)	A closed cycle regenerative gas turbine operating with air as working medium has $P_1 = 1.4$ bar, $T1 = 310$ K, $P_2/P_1 = 5$, $T_{max} = 1050$ K, effectiveness of regenerator = 100%, net work output = 300C kW.
		Assuming the compression and expansion to be isentropic, determine
		(1) Thermal emclency and
		(ii) Mass flow rate of air per minute. (6)
		Or
	(b)	In a gas turbine, the pressure ratio to which air at 15°C is compressed is 6. The same air is then heated to a maximum permissible temperature of 750°C. First in a heat exchanger and then combustion chamber. It is then expanded in two stages such that the expansion work is maximum. The air is reheated to 750°C after the first stage. Determine
		(i) Thermal efficiency of the cycle
		(ii) Work ratio and
		(iii) Net shaft work per kg of air. (5)
		PART C — $(1 \times 15 = 15 \text{ marks})$
16.	(a)	During a test on a Diesel engine, the following observations were made: Power developed by the engine is used for driving a DC Generator. The output of the generator was 210 A at 200 V, efficiency of the generator = 82%. The quantity of fuel supplied to the engine was 11.2 kg/h, calorific value of fuel = 42600 kJ/kg. The air-fuel ratio was 18:1. The exhaust gases were passed through a exhaust gase Calorimeter for which, water circulated through the calorimeter = 580 litres/h, temperature rise of water flowing through the Calorimeter = 36 °C, temperature of exhaust gases at exit of the Calorimeter = 98°C, Heat lost to the jacket cooling water is 32% of total heat supplied. Take ambient temperature = 25°C and Cp of exhaust gase = 1.05 kJ/kg K. Draw the heat balance sheet on minute basis.
	(b)	A 4-cylinder petrol engine has a bore of 57 mm and a stroke of 90 mm. It rated speed is 2800 rpm and it is tested at this speed against a brake which has a torque arm of 0.356 m. The net brake load is 155 N and the fuel consumption is 6.74 liters/h. Specific gravity of fuel used is 0.735 and it has a lower calorific value of 44200 kJ/kg. A Morse test is carried out and the cylinders are cut out in the order 1, 2, 3, 4 with the corresponding brake loads 111, 106.5, 104.2 and 111 N respectively Determine (i) Brake thermal efficiency (ii) specific fuel consumption (iii) Mechanical efficiency
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(iv) Indicated thermal efficiency.